

24-352 Dynamic Systems and Control: QUIZ 2

Close book and notes. You have 45 minutes to complete the following questions:

NAME: _____

12 February 2001

1. Use a free body diagram for the mass shown in Figure 1 to determine its equation of motion. (Note: the displacement $y(t)$ is an "input" that causes the mass to move).
2. Write the equation of motion in terms of an effective stiffness K_e and damping B_e . What are they? What would the natural frequency of the undamped system be if $M = 1$, $K_1 = 1$, $K_2 = 3$, $B_1 = 0.02$, and $B_2 = 0.06$?
3. Suppose $y(t) = Y_0 \sin(\omega t)$. What do we mean by a "steady-state" solution?
4. Find the steady-state solution in terms of general values of M , B_e , K_e , K_2 , B_2 , Y_0 , and ω .

ANSWER

1. $-B_1 \dot{x}$ $-B_2(\dot{x} - \dot{y})$ $-K_1 x$ $-K_2(x - y)$

(5) $M \ddot{x} = -B_1 \dot{x} - K_1 x - B_2(\dot{x} - \dot{y}) - K_2(x - y)$

or $M \ddot{x} + (B_1 + B_2) \dot{x} + (K_1 + K_2) x = B_2 \dot{y} + K_2 y$

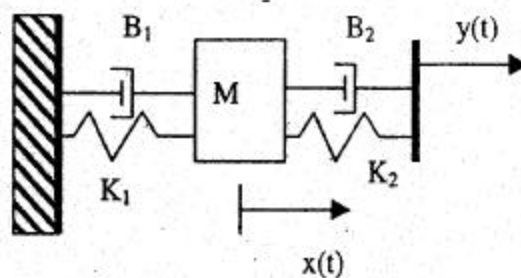


Figure 1

2. $M \ddot{x} + B_e \dot{x} + K_e x = B_2 \dot{y} + K_2 y$, $B_e = B_1 + B_2$, $K_e = K_1 + K_2$
- (3) $\omega_0 = \sqrt{\frac{K_e}{M}} = \sqrt{\frac{4}{1}} = 2 \text{ rad/sec.}$
3. Several possible answers
 - A. Harmonic response with constant amplitude & phase
 - (2) B. The particular solution to the ODE. ✓✓
 - C. The solution after the transient from the initial conditions has died off because of damping. ✓✓
4. See reverse side of paper

Equation of motion: $M\ddot{x} + B_e\dot{x} + K_e x = B_2\dot{y} + K_2 y$ (1)

Assume $y = Y_0 e^{j\omega t}$, $z = A e^{j\omega t}$ where (2)

$M\ddot{z} + B_e\dot{z} + K_e z = B_2\dot{y} + K_2 y$ (3)

Then $x = \text{Im}\{z\}$ (4)

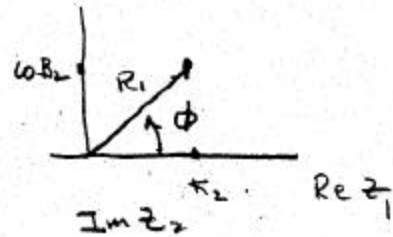
(2) and (3) $\Rightarrow$

$\{K_e M\omega^2 + j\omega B_e\} A e^{j\omega t} = (K_2 + j\omega B_2) Y_0 e^{j\omega t}$

(5) $\Rightarrow A = \frac{Y_0 (K_2 + j\omega B_2)}{K_e M\omega^2 + j\omega B_e}$

Express numerator and denominator as complex exponentials

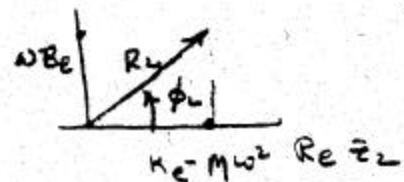
(4) $\left\{ \begin{aligned} z_1 &= K_2 + j\omega B_2 = R_1 e^{i\phi_1} \\ \Rightarrow R_1 &= \sqrt{K_2^2 + \omega^2 B_2^2} \\ \phi_1 &= \tan^{-1}\left(\frac{\omega B_2}{K_2}\right) \end{aligned} \right.$



$z_2 = K_e - M\omega^2 + j\omega B_e = R_2 e^{i\phi_2}$

$R_2 = \sqrt{(K_e - M\omega^2)^2 + \omega^2 B_e^2}$

$\phi_2 = \tan^{-1}\left(\frac{\omega B_e}{K_e - M\omega^2}\right)$



(2) $\left\{ \begin{aligned} \text{Then } z &= \frac{Y_0 R_1}{R_2} e^{j(\omega t + \phi_1 - \phi_2)} \end{aligned} \right.$

and $x = \text{Re}\{z\} = \frac{Y_0 R_1}{R_2} \sin(\omega t + \phi_1 - \phi_2)$