

ME 24-221
THERMODYNAMICS I

Solution to Exam 2
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Solution to Pb.1:

Given: Refrigerator

$$T_L = -5^\circ\text{C} = 268.15 \text{ K}$$
$$T_H = 27^\circ\text{C} = 300.15 \text{ K}$$
$$\beta_{\text{claimed}} = 10$$

β_{carnot} is the maximum C.O.P one can get.

$$\beta_{\text{carnot}} = \frac{T_L}{T_H - T_L} = \frac{268.15}{32} = 8.38$$

$\beta_{\text{claimed}} > \beta_{\text{carnot}}$ which is not possible.

Hence the claim is **impossible**. -----(a)

$$\beta = \frac{Q_L}{W} : \text{Hence, } W = \frac{Q_L}{\beta} \text{ where } Q_L = 2 \text{ kW}$$

W is minimum when β is maximum i.e., β_{carnot} .

$$\text{Hence } W_{\text{min}} = \frac{Q_L}{\beta_{\text{carnot}}} = \frac{2}{8.38} = 0.238 \text{ kW} \text{ -----(b)}$$

Solution to Pb 2:

Given: Water

$$\text{State 1: } T_1 = 1073.15 \text{ K; } P_1 = 5000 \text{ kPa; } \dot{m}_1 = 10 \text{ kg/s}$$

$$\text{State 2: } T_2 = 773.15 \text{ K; } P_2 = 100 \text{ kPa; } \dot{m}_2 = 10 \text{ kg/s}$$

$$\text{State 3: } P_3 = 30 \text{ kPa; } x_3 = 0.9 \text{ (Saturated; pressure entry; Table B.1.2)}$$

States 1 and 2 are superheated. Hence from Table B.1.3, $h_1 = 4137.17 \text{ kJ/kg}$ and
 $h_2 = 3478.44 \text{ kJ/kg}$

$$\text{Continuity: } \dot{m}_1 + \dot{m}_2 = \dot{m}_3 = 15 \text{ kg/s}$$

$$\dot{V}_3 = \frac{\dot{m}_3}{\rho_3} = v_3 \dot{m}_3$$

$$v_3 = v_{f,3} + x_3 v_{fg,3} = 0.001022 + (0.9)5.22816 = 4.706366 \text{ m}^3/\text{s}$$

$$\text{Therefore, } \dot{V}_3 = 15 * 4.706366 = 70.59 \text{ m}^3 \text{ -----(a)}$$

First Law for CV, SSSF: $\dot{Q}_{CV} = 0$ (adiabatic; also neglecting KE and PE changes)

$$\dot{m}_3 h_3 - \dot{m}_1 h_1 - \dot{m}_2 h_2 = -\dot{W}_{CV}$$

$$\text{And, } h_3 = h_{f,3} + x_3 h_{fg,3} = 289.21 + (0.9)2336.07 = 2391.673 \text{ kJ/kg}$$

$$\text{Hence, } \dot{W}_{CV} = 10(4137.17) + 5(3478.44) - 15(239.673) = 22888.80 \text{ kW -----(b)}$$

Solution to Pb. 3:

Given: Air

$$\text{State 1: } T_1 = 473.15 \text{ K; } P_1 = 500 \text{ kPa; } \dot{m}_1 = 2 \text{ kg/s}$$

$$\text{State 2: } T_2 = 373.15 \text{ K; } P_2 = 500 \text{ kPa; } \dot{m}_2 = 5 \text{ kg/s}$$

$$\text{State 3: } P_3 = 100 \text{ kPa; } V_3 = 50 \text{ m/s}$$

$$\dot{Q}_{CV} = 20 \text{ kW}; \dot{W}_{CV} = 40 \text{ kW}$$

First Law for CV, SSSF: (Neglecting the inlet Velocities and PE change)

$$\dot{m}_3 \left(h_3 + \frac{V_3^2}{2} \right) - \dot{m}_1 h_1 - \dot{m}_2 h_2 = \dot{Q}_{CV} - \dot{W}_{CV}$$

$$T_{\text{ref}} \text{ as } 0\text{K, } h_3 = C_{po} T_3; \quad h_1 = C_{po} T_1 = 1.004(473.15) = 475.04 \text{ kJ/kg};$$

$$h_2 = C_{po} T_2 = 1.004(373.15) = 374.64 \text{ kJ/kg};$$

$$\text{Also, for consistency of units: } \frac{V_3^2}{2} = \frac{50^2}{2 \times 1000} = 1.25 \text{ kJ/kg}$$

$$\text{i.e., } 7(C_{po} T_3 + 1.25) - 2(475.04) - 5(374.63) = 20 - 40$$

$$7C_{po} T_3 + 8.75 - 2823.28 = -20$$

$$T_3 = 397.63 \text{ K (or } 124.48^\circ\text{C)} \text{ -----(a)}$$

$$\text{Diameter } D = \sqrt{\frac{4A}{\pi}}$$

$$A_3 = \frac{\dot{V}_3}{V_3} = \frac{\dot{m}_3 v_3}{V_3}$$

Since for air, $P_3 v_3 = RT_3$ we get, $v_3 = RT_3/P_3 = (0.287)(397.63)/100 = 1.1412 \text{ m}^3/\text{kg}$

$$A_3 = \frac{\dot{m}_3 v_3}{V_3} = \frac{7 \times 1.1412}{50} = 0.1597 \text{ m}^2$$

$$D_3 = 0.45 \text{ m} \text{ -----(b)}$$

$$\begin{aligned} \dot{m}_3 u_3 - \dot{m}_1 u_1 - \dot{m}_2 u_2 &= 7C_{vo}T_3 - 2C_{vo}T_1 - 5C_{vo}T_2 \\ &= (7 \times 0.717 \times 397.63) - (2 \times 0.717 \times 473.15) - (5 \times 0.717 \times 373.15) \\ &= -20.53 \text{ KW} \text{ -----(c)} \end{aligned}$$